

Amendments to the Specification:

Please amend the specification as follows:

Please add the following heading before page 1, line 4 of the application as filed:

BACKGROUND

Please add the following heading before page 2, line 26 of the application as filed:

SUMMARY

Please add the following heading before page 7, line 8 of the application as filed:

BRIEF DESCRIPTION OF THE DRAWINGS

Please add the following heading before page 7, line 22 of the application as filed:

DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENTS

Please replace page 5, lines 20-23 of the application as filed with the following paragraph:

In an advantageous embodiment, the threshold value of the intake pressure is set at **at** least 3 bar, preferably 5 bar, below the value of the saturation pressure caused by the ambient temperature.

Please replace page 5, lines 25-29 of the application as filed with the following paragraph:

Alternatively, the circuit can also be operated for a predeterminable period of time in the cooling mode or in the laevorotatory triangulation process, the circuit likewise being changed over to the heating mode again after the ~~expiry~~ **expiration** of the period of time.

Please replace page 6, line 37 – page 7, line 6 of the application as filed with the following paragraph:

In a further advantageous embodiment, the condenser is connected to the evaporator on the exit side via a control means and on the entry side via an associated controllable connecting line, after the opening of the control means gaseous fluid ~~passing~~ **passes** into the

evaporator and ~~forcing~~ forces the liquid fluid located in the evaporator out of the evaporator into the active part of the circuit.

Please replace page 10, lines 5-29 of the application as filed with the following paragraph:

In order, as illustrated in figure 1, to avoid additional components for the air conditioning system 1, a suction pressure of the condenser 26 is in this case ~~[[is]]~~ set in such ~~[[as]]~~ a way that the suction pressure at least partially overshoots a saturation pressure caused by the ambient temperature. The setting of the suction pressure is in this case brought about in a particularly simple way by means of structural features of the components of the air conditioning system 1. For example, for this purpose, a storage or evaporator volume representing the evaporator 6 is designed to be so small that the fluid quantity or refrigerant quantity collected or stored in the intermediate store 28 (= header) cannot condense completely in the cold evaporator 6, the shutoff device 12, for example an expansion valve, preventing a further outflow into the likewise cold gas cooler 32. Alternatively, the intermediate store 28 may have a correspondingly large storage volume which is substantially larger than the evaporator volume, the volume of the evaporator lying, for example, in the range of 50 to 500 ccm and the volume of the header lying in the range of 200 to 2000 ccm, so that a ratio of the volume of the header to the volume of the evaporator in the range of 2:1 to 20:1, preferably 2:1 to 10:1, can be selected.